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(54) A double-bevel spring retaining ring.

(57) A double-bevel spring retaining ring of the type adapted for assembly within a groove provided therefor in a housing bore or on a shaft to form an artificial shoulder for axially locating a machine part in the bore or on the shaft. The spring comprises an open-ended ring body of spring material. The body includes opposite side surfaces, each surface including a generally radial shoulder-forming portion and a groove-seating portion (30B) which is inclined at an acute angle relative to the shoulder forming portion. Either of the groove-seating surface portions (30B) are adapted to engage a correspondingly inclined wall of the groove to take-up axial play of the machine part. The length of each groove seating surface portion (34-M) is from 80% to 120% of the minimum groove penetration depth (32-M) for the ring.

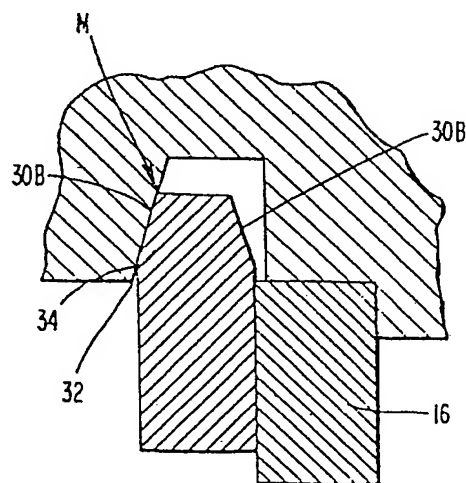


Fig. 8

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A DOUBLE-BEVEL SPRING RETAINING RING

The present invention relates to a double-bevel spring retaining ring of the type adapted for assembly within a groove to form an artificial shoulder for axially locating a machine part.

Split retaining rings are employed in assemblies, such as machines for example, to retain various components such as pulleys, sleeves, bearings, etc., against axial displacement. Examples of such rings may be found in U.S. Patent No. 2,509,081 issued to Bluth on May 23, 1950, U.S. Patent No. 2,544,631 issued to Heimann on March 6, 1951, U.S. Patent No. 2,861,824 issued to Wurzel on November 25, 1958, and German Patent No. 767,134. Such rings are formed of spring material so that they can be compressed for insertion into an internal groove, or expanded for insertion into an external groove, of the machine, whereupon the ring springs-back into contact with the groove. The ring projects beyond the groove and forms an artificial stop shoulder to axially retain the particular assembly component.

In an effort to minimize axial looseness or end-play of the assembly component, split retaining rings have been heretofore provided with a beveled side surface

portion which engages a correspondingly inclined wall of the groove. Accordingly, the spring action of the ring within the groove results in the ring being wedged in an axial direction to press the assembly component against another component or shoulder of the machine and thereby eliminate end-play.

One problem which has resulted from the use of beveled rings relates to the possibility of the ring being inserted backwards such that the beveled surface engages the retained machine component rather than the inclined groove wall. A backwardly mounted beveled ring is unable to restrain appreciable axial loads and is too easily dislodged from the groove when acted upon by the retained component.

One previously proposed solution to the problem of backwardly mounted rings involves the provision of bevels on both sides of the ring so that either orientation of the ring within the groove is proper. While overcoming the misassembly problem, such a double-bevel ring has exhibited little resistance to axial thrust loads and tends to be dislodged from the groove much more easily than a single bevel ring. Accordingly, double-bevel rings have not been commercially successful.

It is, therefore, an object of the present invention to minimize or obviate problems of the type previously discussed, by providing a double-bevel spring retaining ring which can take-up end play and yet exhibits a high resistance to axial thrust loads.

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According to the present invention there is provided a double-bevel spring retaining ring of the type adapted for assembly within a groove to form an artificial shoulder for axially locating a machine part, said ring comprising an
5 open-ended ring body of spring material, which body has opposite side surfaces each including a generally radial shoulder-forming portion and a groove-seating portion which is inclined at an acute angle relative to the shoulder-forming portion, either of said groove-seating portions
10 being adapted to engage a correspondingly inclined wall of the groove to take-up axial play of the machine part, characterised in that the length of each groove-seating surface portion is from 80% to 120% of the minimum groove penetration depth for said ring.

15 Some embodiments of the invention will now be described, by way of example, with reference to the accompanying drawings in which like numerals designate like elements, and in which:

Figure 1 is a front view of a prior art retaining
20 ring of the type having a tapered cross-section;

Figure 2 is a fragmentary longitudinal sectional view of the ring of Figure 1 seated within a groove in an assembly;

Figure 3 is a view similar to Figure 2 depicting
25 the ring in a backward orientation;

Figure 4 is a view similar to Figure 2 of a prior art double-bevel ring;

Figures 5 and 6 are cross-sectional views through different portions of a tapered cross-section, double-bevel ring according to the present invention, with prior art bevel surfaces being indicated by phantom lines;

5 Figure 7 is a fragmentary cross-sectional view of one form of double-bevel according to the present invention disposed within a groove;

Figure 8 is a view similar to Figure 7 of another form of double-bevel according to the present invention;

10 Figure 9 is a view similar to Figure 7 of still another form of double-bevel according to the present invention; and

Figures 10 and 14 are front views of different types of rings containing a double-bevel according to the
15 present invention.

Figures 1 and 2 depict a conventional type of split spring retaining ring 10 which has a varying or tapering cross-section, and a bevel 12 on one side of an outer, groove-seating edge of the ring. During use, the
20 ring 10 is installed within an annular groove 13 of an assembly 14 to retain a component 16 of the assembly. The bevel surface 12 engages an inclined wall 18 of the groove whereby the ring 10 is wedged in an axial direction 19 to press the component 16 firmly against a wall 20 of the
25 assembly. In this manner, axial looseness or end play of the component is eliminated.

Generally, the bevel 12 is inclined at a 10 to 20 degree angle from the non-inclined radial portion 15 of the same side of the ring. Angles much larger than that do


not provide a secure seating of the ring, while smaller angles do not provide ample end play take-up.

For each split spring retaining ring there exists a minimum desirable depth of groove penetration by the ring in order to assure that a secure seating of the ring in the groove is established. For a ring of tapered cross-section, for example, as depicted in Figures 1-2, such minimum penetration depth from point 24 to point M is one-half of the groove depth D measured along the inclined wall 18. For other types of rings such minimum penetration depth may differ, as by being one-third, one-fourth, etc., of the groove depth D. Of course, due to its spring characteristics, the ring 10, following initial insertion, attempts to penetrate the groove yet more deeply until either all axial play is taken-up, or the ring bottoms-out against the base 17 of the groove. Thus, the standard practice in the art has been to dimension the length of the bevel surface 12 greater than the length of the inclined wall 18 so that the bevel surface protrudes slightly beyond the groove even when the ring has bottomed-out, thereby assuring full contact of the wall 18 by the surface 12. Stated another way, the length of the bevel portion is over one-hundred percent greater than the minimum groove penetration depth $D/2$. That is, the distance between points 25 and 24 in Figure 2 is more than the distance between points 24 and M, the latter representing the minimum groove penetration depth for that particular ring. (For rings

whose minimum penetration depth is $D/3$ the beveled portion is over two-hundred percent longer than the minimum penetration depth, and over three-hundred percent longer for rings whose minimum penetration depth is $D/4$.)

5 In Figure 3, there is depicted a situation in which a single-bevel ring 10 is accidentally inserted in a backwards manner within the groove 13. In such a case the ring barely penetrates the groove and is thus highly unstable and easily dislodged in response to forces applied
10 by the retained component 16 in the axial direction 22.

In Figure 4 there is depicted a split retaining ring 10A containing a double bevel 12A, 12B in the manner heretofore proposed in the art. Such bevels are designed in accordance with previously followed principles in the
15 art, viz., the length of each bevel surface being over 100 percent longer than the minimum groove penetration depth. In practice, however, it has been found that such a double bevel ring 10A exhibits surprisingly low load-carrying capacity and is easily dislodged in a direction indicated
20 by the broken-line position of the ring 10A in Figure 4. That is, the previously proposed double-bevel rings are adapted to be easily swung about a pivot 24 defined by the outer edge of the inclined wall 18 of the groove 13, in response to urgings from the retained component 16. In a
25 single-bevel ring as depicted in Figure 2, such swinging movement is resisted by the non-beveled portion 26 of the ring located within the groove and immediately adjacent the component 16 which contacts the ring.



The absence of such a ring portion in the previously proposed double-bevel rings, however, renders the rings unstable. In addition, the absence of such a ring portion results in an increase in distance G between point 24 and the resultant F of the axial dislodging force applied by the retained component 16. That is, in the single-bevel ring of Figure 2, the distance G', which constitutes the moment arm for the ring-dislodging force F' is shorter, resulting in less of a moment than in the double-bevel ring of Figure 4.

As a result, the use of double-bevel rings has not been practicable, resulting in a continuance of the problem involving backwards insertion of the rings (Fig. 3).

In accordance with the present invention, a double-bevel spring retaining ring is provided which is secure when installed within the groove in a housing bore or shaft or the like, and exhibits ample load-carrying capacity. The invention is based upon the discovery that such characteristics are achieved by a spring retaining ring having a double-bevel, with each bevel surface having a length which does not exceed the minimum groove penetration depth by more than 20% of the latter, or is no shorter than the minimum groove penetration depth by more than 20% of the latter.

In Figures 5 and 6, two cross-sectional views are depicted through different locations of a tapered section height retaining ring 28 having a double-bevel 30 according to the present invention. Depicted in broken lines, for

comparison purposes, are the previously proposed double-bevels 12A, 12B. The prior art bevel surfaces are dimensioned to be over 100% longer than the minimum penetration depth, whereas the bevel surfaces according to
5 the present invention are dimensioned so that any variance with the minimum penetration depth lies within the range \pm 20%, i.e., the length of each groove seating surface on the ring is from 80% to 120% of the minimum penetration depth.

10 In keeping with the principles of the present invention, some particular examples of the double-bevel will now be described.

In Figure 7 an embodiment of the present invention is depicted wherein the length of the bevel surface 30A
15 is equal to the minimum groove penetration, which is the distance from point 31 to point M.

In Figure 8 an embodiment of the present invention is shown wherein the length of the bevel surface 30B is shorter than the minimum penetration depth by 20% of the
20 latter. That is, the distance between points 32 and 34 in Figure 8 is 20% of the distance between points 32 and M.

In Figure 9 there is illustrated an embodiment of the invention wherein the length of the bevel surface 30C exceeds the minimum penetration depth by 20% of the latter.
25 That is, the distance between points 36 and 38 is 20% of the distance between points 38 and M.

During tests of rings configured in accordance with Figures 7-9, it was found that the rings remain

adequately secure within the groove and exhibit sufficient load-carrying capacity to resist loads exerted by the retained component 16.

5 The double-bevel concept according to the present invention can be employed in connection with any kind of split spring retaining rings such as both internal and external types of such rings. Examples are shown in Figures 10 to 14. In Figures 10 and 11 retaining rings 40, 42 are depicted which are of the uniform cross-section type.
10 The ring 40 is provided with a double-bevel 40A at the outer edge, whereas the ring 42 is provided with a double-bevel 42A at an inner edge.

In Figures 12 and 13 retaining rings 44, 46 of the multi-truss type are depicted. Rings of that type are
15 described for example in U.S. Patent No. 4,006,659 issued to Wurzel et al on February 8, 1977. The ring 44 contains a double-bevel 44A at an inner edge thereof, whereas the ring 46 contains a double-bevel, at an outer edge 46A thereof.

In Figure 14 a retaining ring 48 of the variable
20 cross-section type is depicted. That ring contains a double-bevel 48A at an inner edge thereof.

As a result of the present invention, problems involving backwardly inserted rings have been eliminated by a ring which exhibits ample load-bearing capacity.

CLAIMS:

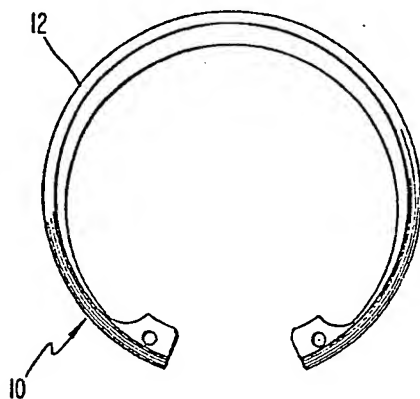
1. A double-bevel spring retaining ring of the type adapted for assembly within a groove to form an artificial shoulder for axially locating a machine part, said ring comprising an open-ended ring body of spring material, which body has opposite side surfaces each including a generally radial shoulder-forming portion and a groove-seating portion which is inclined at an acute angle relative to the shoulder-forming portion, either of said groove-seating portions being adapted to engage a correspondingly inclined wall of the groove to take-up axial play of the machine part, characterised in that the length of each groove-seating surface portion is from 80% to 120% of the minimum groove penetration depth for said ring.

2. A retaining ring according to Claim 1, wherein said inclined groove-seating surface portions of said ring are located at a radially inner edge of said ring.

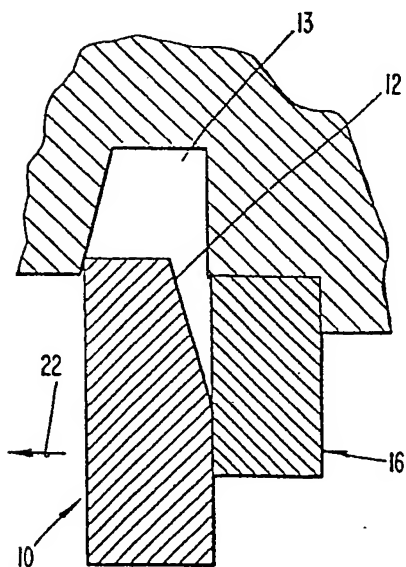
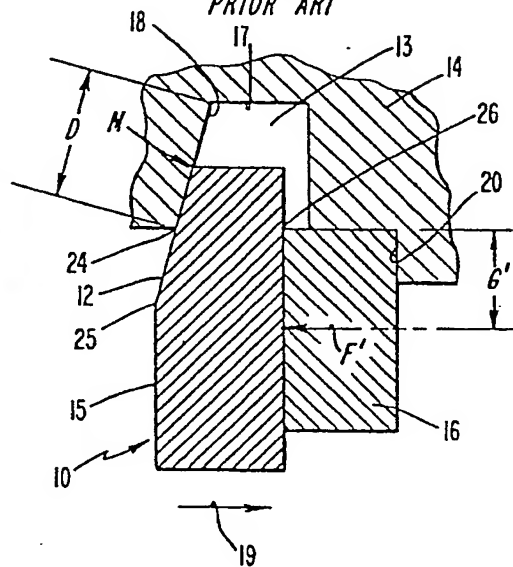
3. A retaining ring according to Claim 1, wherein said inclined groove-seating surface portions of said ring are located at a radially outer edge of said ring.

Fig. 1

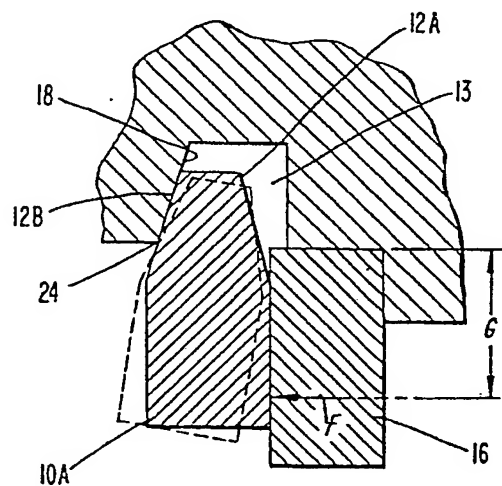
PRIOR ART

*Fig. 2*

PRIOR ART

*Fig. 3*

PRIOR ART

*Fig. 4*

PRIOR ART

Fig. 5

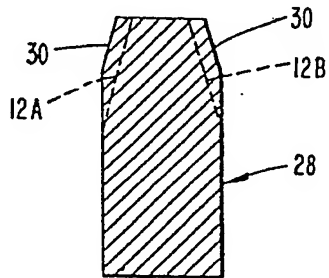


Fig. 6

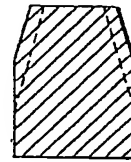


Fig. 7

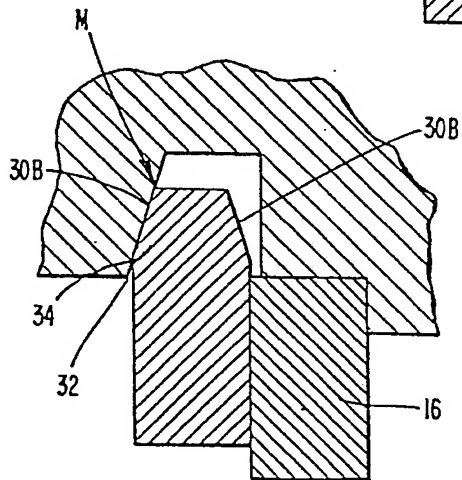
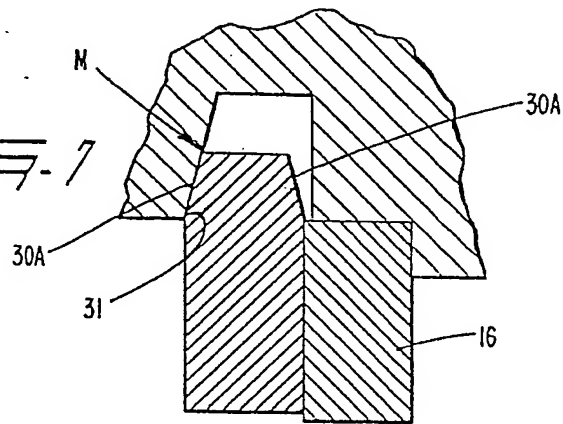


Fig. 8

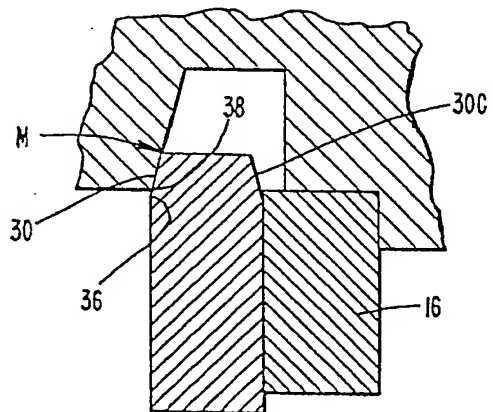


Fig. 9

FIG. 10

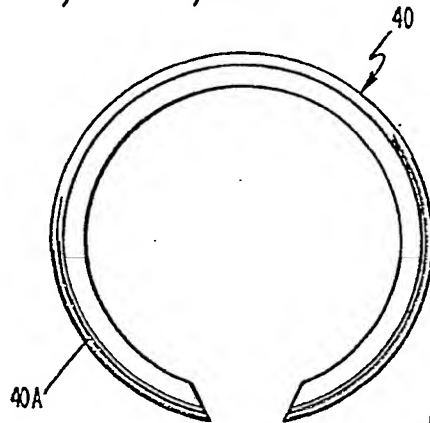


FIG. 11

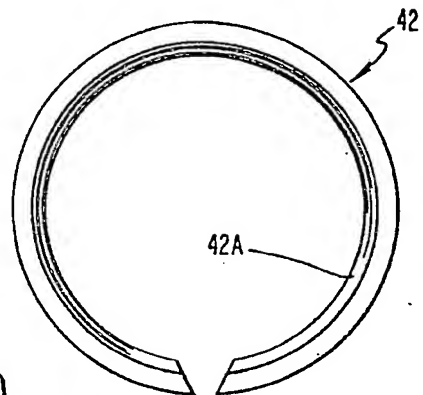


FIG. 12

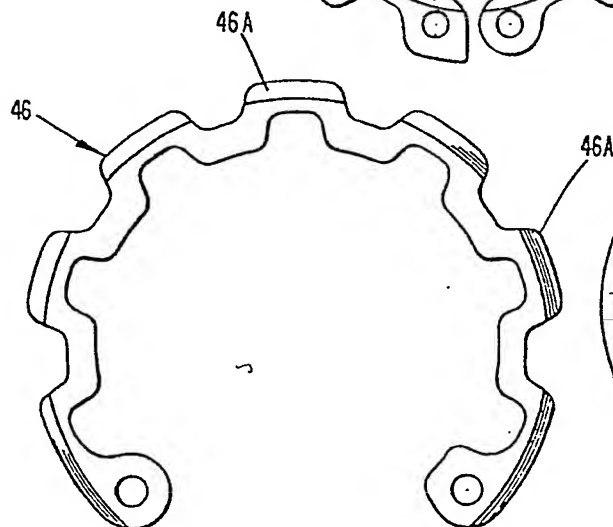
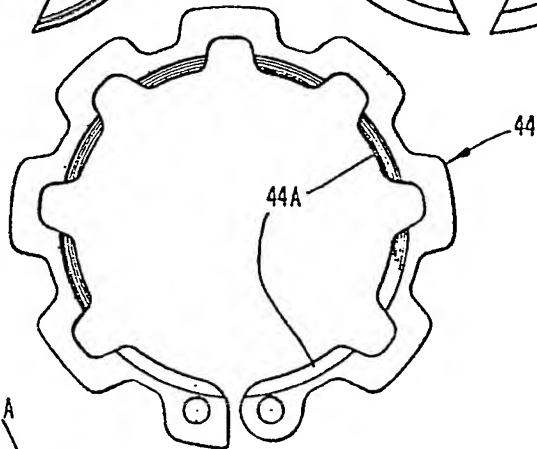


FIG. 13

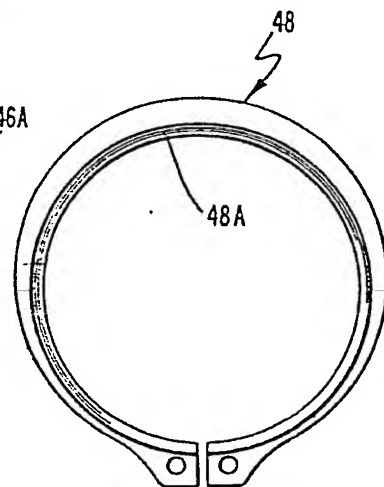
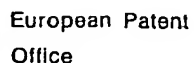


FIG. 14



Application number

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